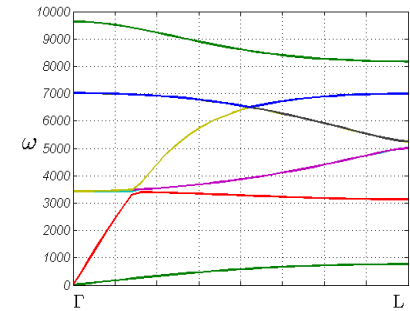
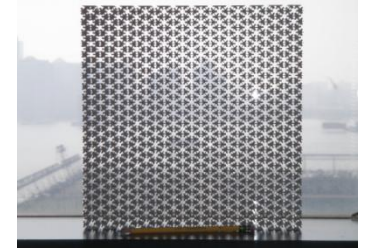
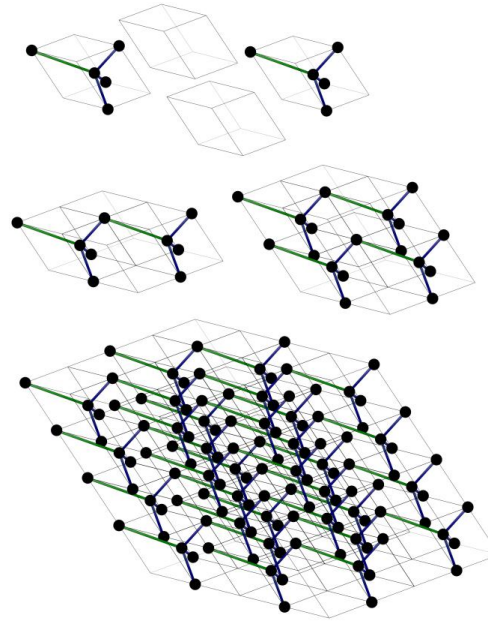


# Dynamics of pentamode structures using beam theory

Andrew N. Norris  
Adam J. Nagy  
Xiaoshi Su

Anton A. Kutsenko (U. Bordeaux)



*AmeriMech 2014 (2 - 4 April 2014)*

## Dynamics of periodic materials and structures

Georgia Tech Hotel and Conference Center, Atlanta, Georgia, USA

*Sponsors*



## **pentamode materials**

pentamode lattice: statics

pentamode lattice: dynamics



Kelvin (1856)

$$C_{ijkl} = \sum_{\alpha=1}^6 K_{\alpha} P_{ij}^{\alpha} P_{kl}^{\alpha} \quad \rightarrow \quad \mathbf{C} = \sum_{\alpha=1}^6 K_{\alpha} \mathbf{P}^{\alpha} \otimes \mathbf{P}^{\alpha}$$

Positive definite strain energy :  $K_{\alpha} > 0$



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Positive definite strain energy :  $K_{\alpha} > 0$



Maxwell (1864)

Necessary and sufficient conditions for truss to be rigid



Necessary and sufficient condition for rigidity of 2D and 3D frameworks is  $Z > 6$  and  $Z > 12$ , respectively  
Deshpande et al. *JMPS* (2001).



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Kelvin (1856)

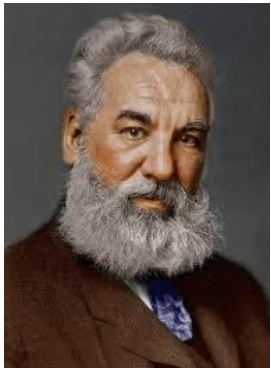
Positive definite strain energy :  $K_{\alpha} > 0$

Necessary and sufficient conditions for truss to be rigid



Necessary and sufficient condition for rigidity of 2D and 3D frameworks is  $Z > 5$  and  $Z > 11$ , respectively  
Deshpande et al. *JMPS* (2001).

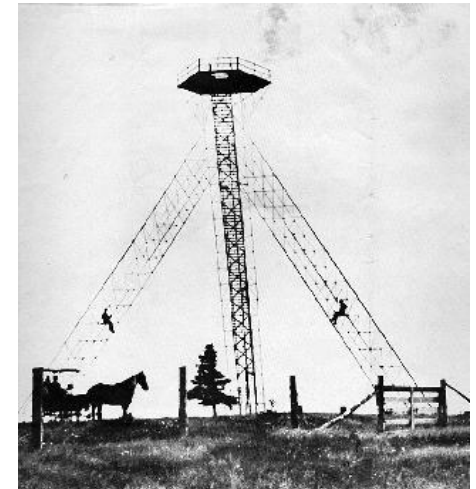
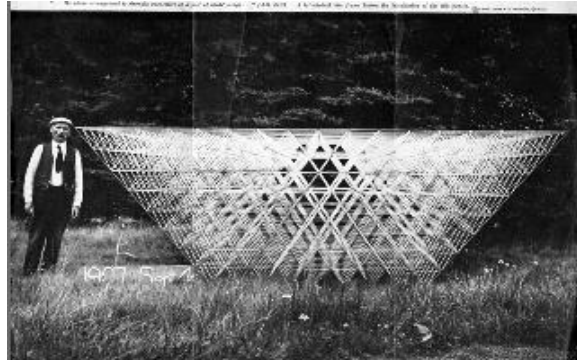
Maxwell(1864)



**A. Graham Bell**  
**Has New Idea**  
**In Architecture**

Opening of the Tetrahedral Tower, Seventy Feet High on Beinn Bhreagh.

It May Become an Important Factor in Building of the Future.



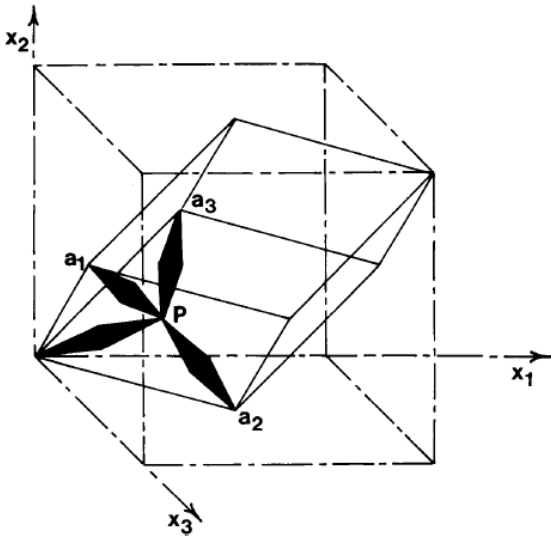
Bell (1907)

Kelvin

$$C_{ijkl} = \sum_{\alpha=1}^6 K_{\alpha} P_{ij}^{\alpha} P_{kl}^{\alpha} \quad \rightarrow \quad \mathbf{C} = \sum_{\alpha=1}^6 K_{\alpha} \mathbf{P}^{\alpha} \otimes \mathbf{P}^{\alpha}$$

$$K_{\alpha} > 0, \quad \alpha = 1, 2, \dots, 6$$

## Milton and Cherkaev (1995)



proposed diamond structure

$$K_1 = 0, \quad K_{\alpha} > 0, \quad \alpha = 2, \dots, 6$$

Unimode

$$K_1 = K_2 = 0,$$

Bimode

.....

.....

$$K_1 = \dots = K_5 = 0, \quad K_6 > 0$$

**Pentamode (PM)**

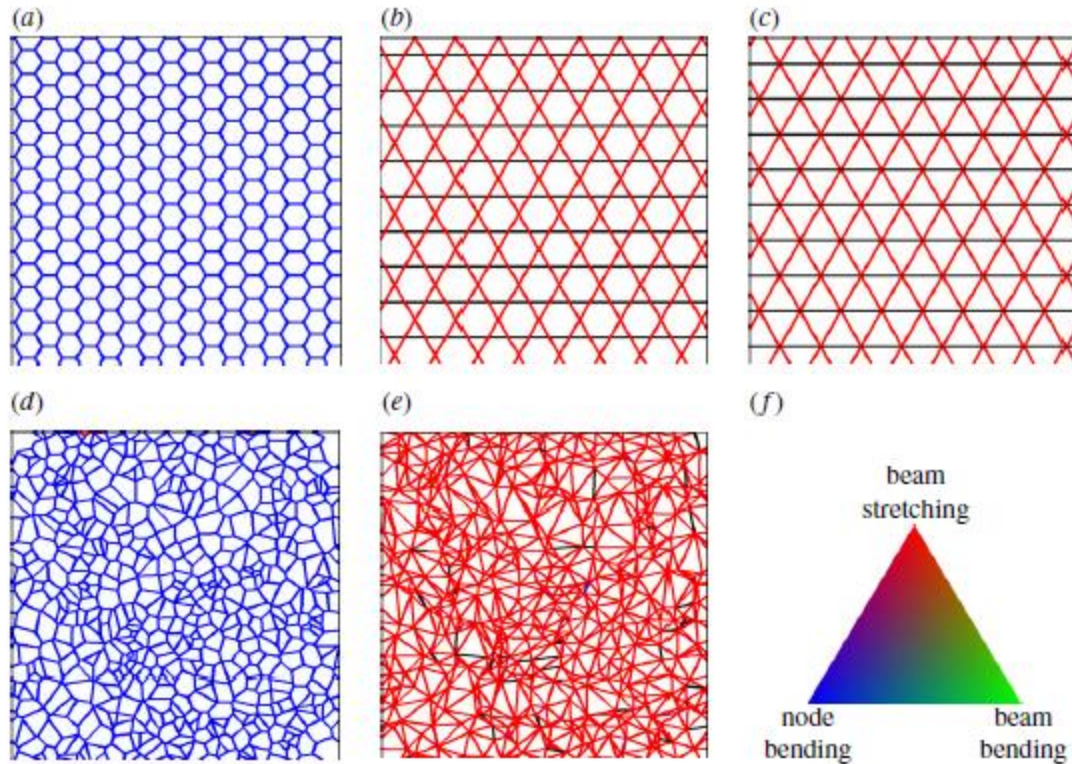
**PM:** five of the eigen-stiffnesses are zero

five (*penta*) easy/soft modes

Warren & Kraynik (1988, ..) Ashby, Deshpande, Hutchinson & Fleck, Christensen (1995, 2000)

Stretch dominated, bending dominated, collapse mechanisms, ....  
.....soft modes, easy modes, isostatic

Stretch dominated, bending dominated, collapse mechanisms, ....  
.....soft modes, easy modes, isostatic

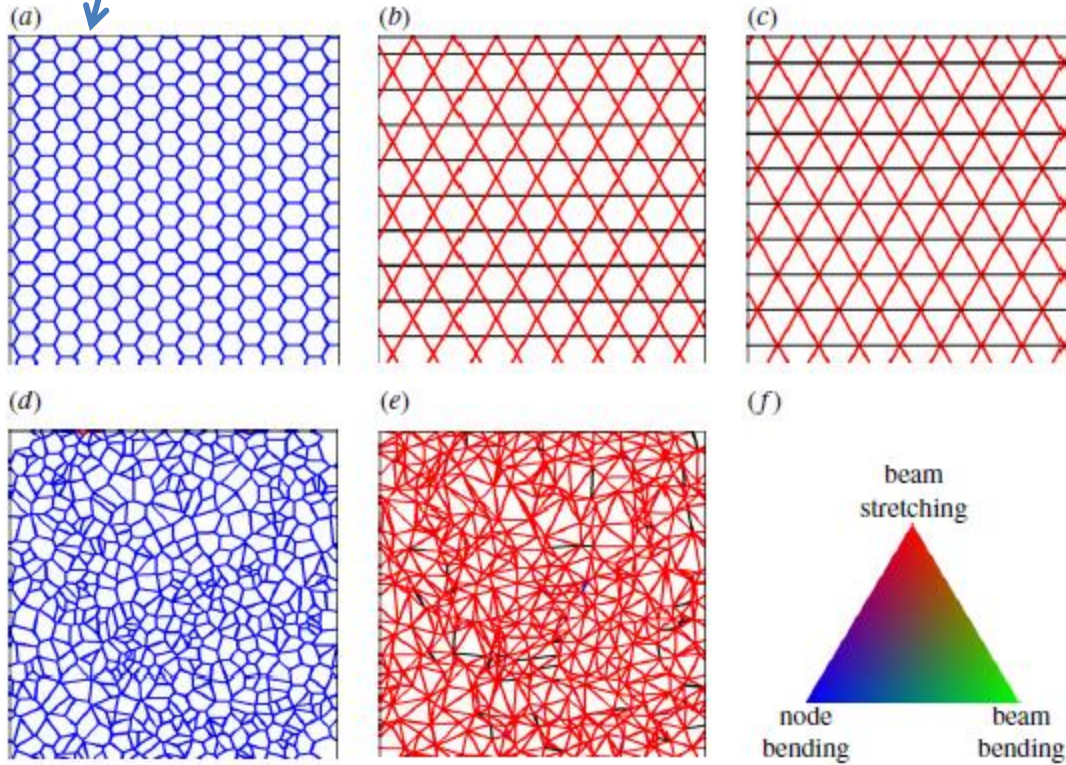




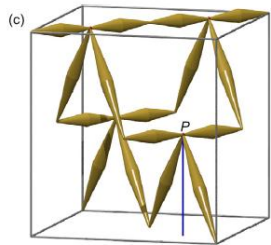
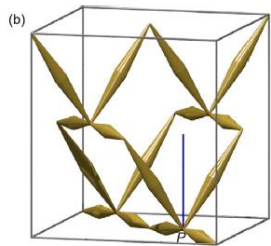
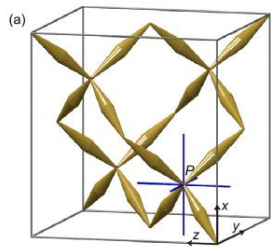
soft modes are bending dominated

$K_\alpha > 0$  i.e. effective static moduli, are stretch dominated

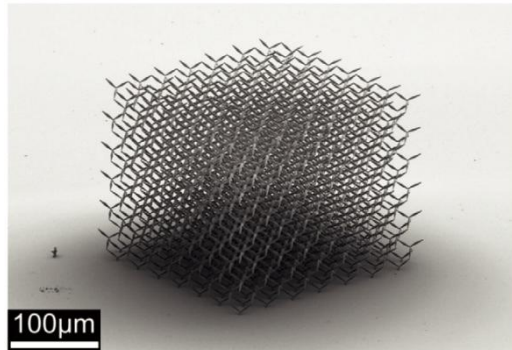
soft/easy  
modes



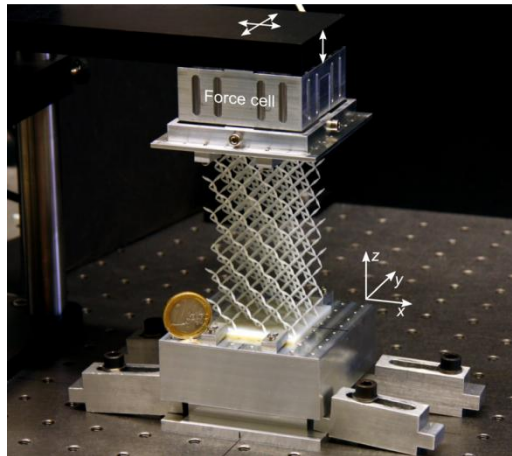
# Pentamode lattice structures



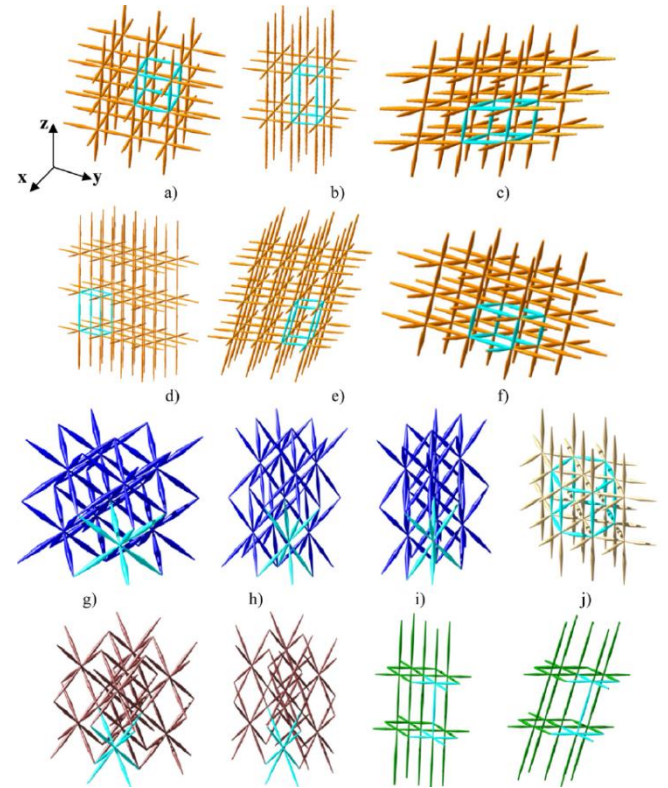
Kadic et al.,  
*NJP* 2013



Kadic et al., *APL* 2012



Schnitty et al., *APL* 2013



Mejica and Lantada *Smart Mat. Struct.* 2013

# water as an elastic “solid”

elastic equation of motion  $\text{div} \boldsymbol{\sigma} = \rho \ddot{\mathbf{u}}$

$$\sigma_{ij} = C_{ijkl} \varepsilon_{kl} \quad \rightarrow \quad \boldsymbol{\sigma} = \mathbf{C} \boldsymbol{\varepsilon}$$

Acoustics  $C_{ijkl} = K \delta_{ij} \delta_{kl} \quad \rightarrow \quad \mathbf{C} = K \mathbf{I} \otimes \mathbf{I}$

Water is a **pentamodal elastic material**

**transformation acoustics:** isotropic PM  $\longrightarrow$  anisotropic PM  
Norris (2008, 2009)

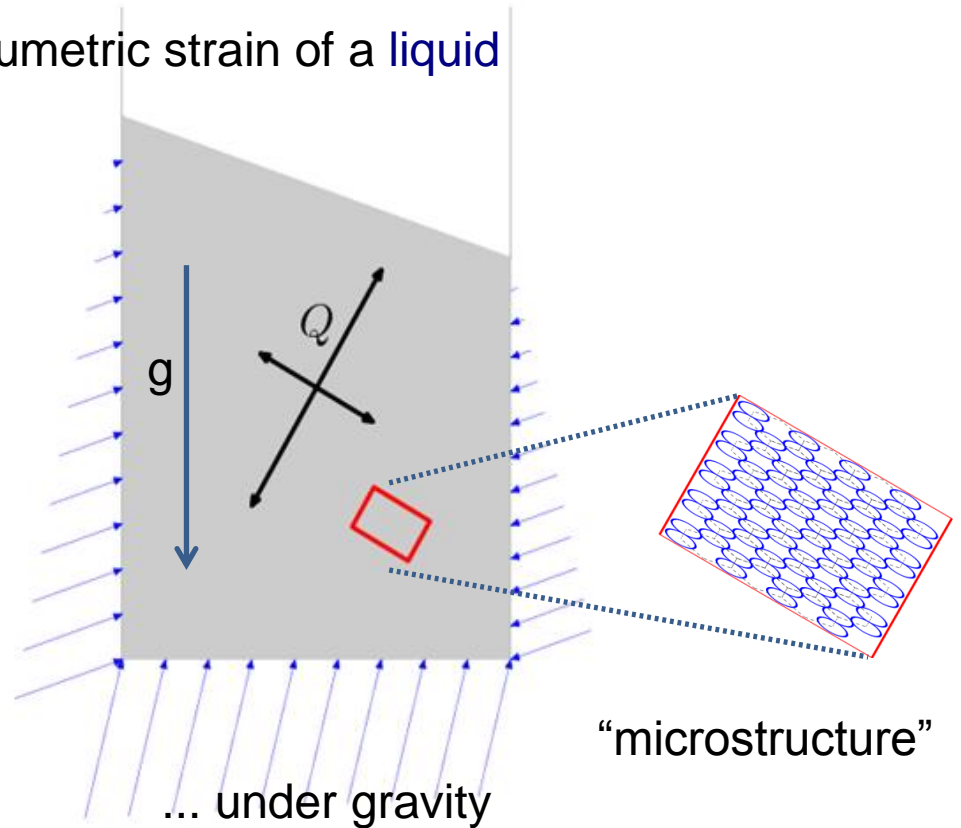
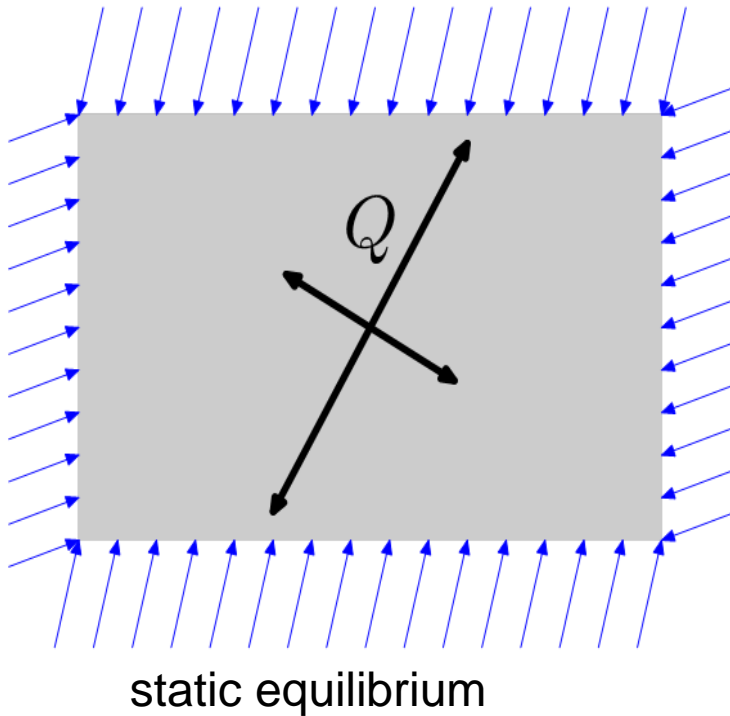
**pentamode** form of stiffness:

$$\mathbf{C} = K \mathbf{S} \otimes \mathbf{S}$$

mechanical behavior of **pentamode materials (PM)**  $C_{ijkl} = K Q_{ij} Q_{kl}$

a single type of stress (and strain)

- like hydrostatic stress and volumetric strain of a liquid

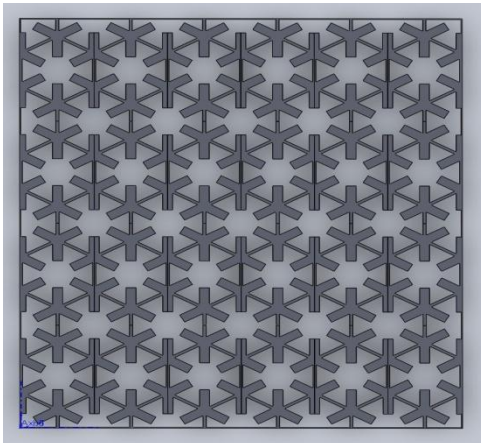
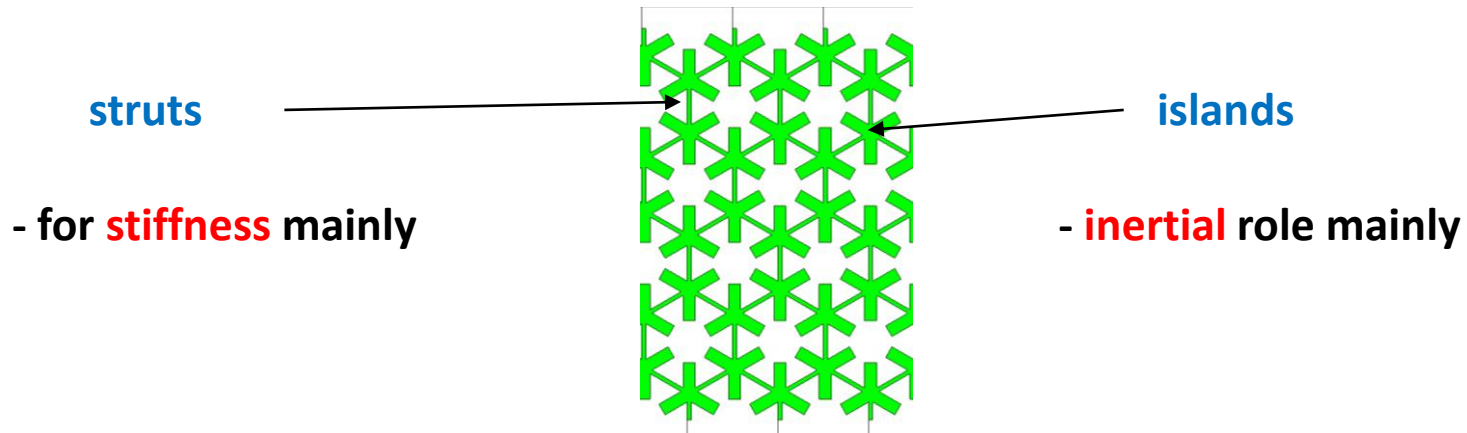


**PM** = limiting case of anisotropic solids with zero "shear" rigidity

# Metal Water

generic structure for **transformation acoustics** in water

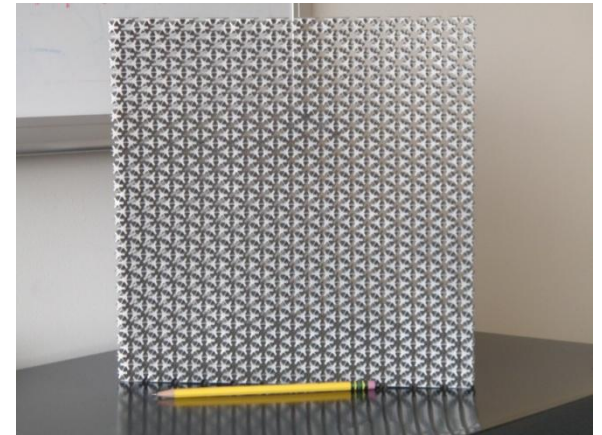
Norris, Nagy (2011)



bulk modulus = 2.25 Gpa

density = 1000 kg/m<sup>3</sup>

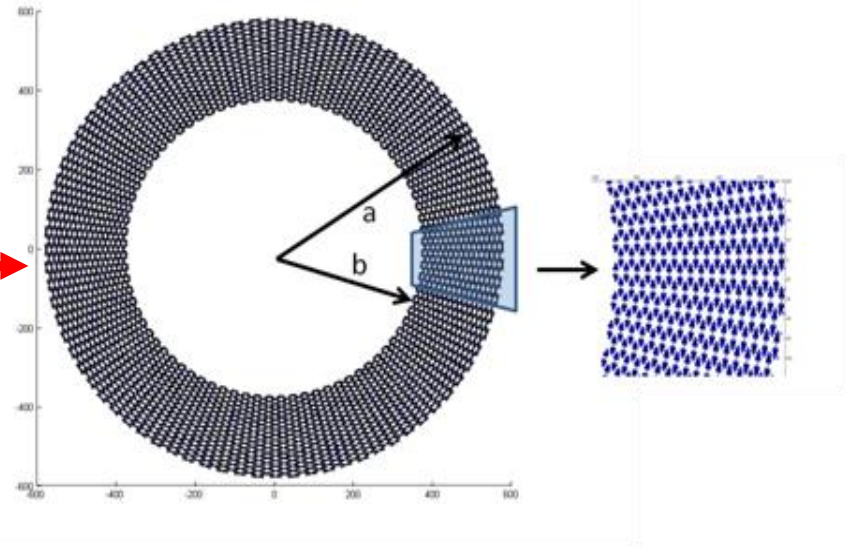
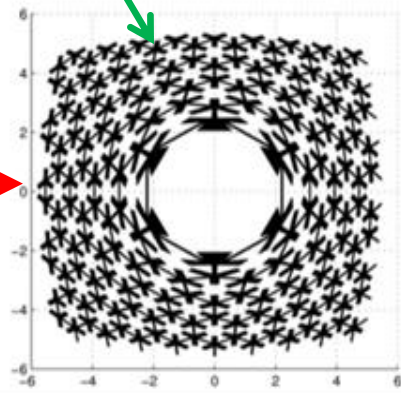
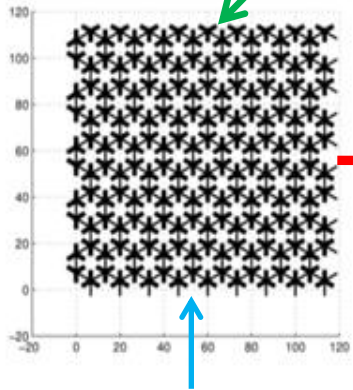
shear modulus = 0.065 Gpa  
(i.e. small)



# Pentamode material and transformation acoustics

same amount of total empty (cloaked) space

heavy metal preferred



voids "invisible"

volume of empty space remains constant

conservation of empty/cloaked space = conservation of mass

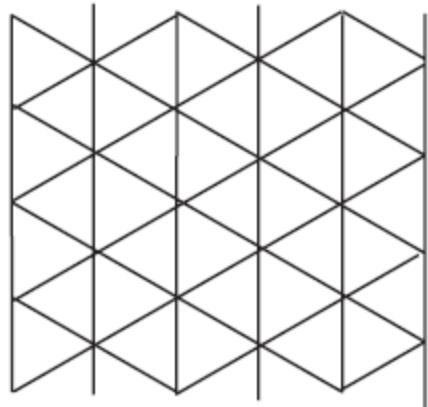
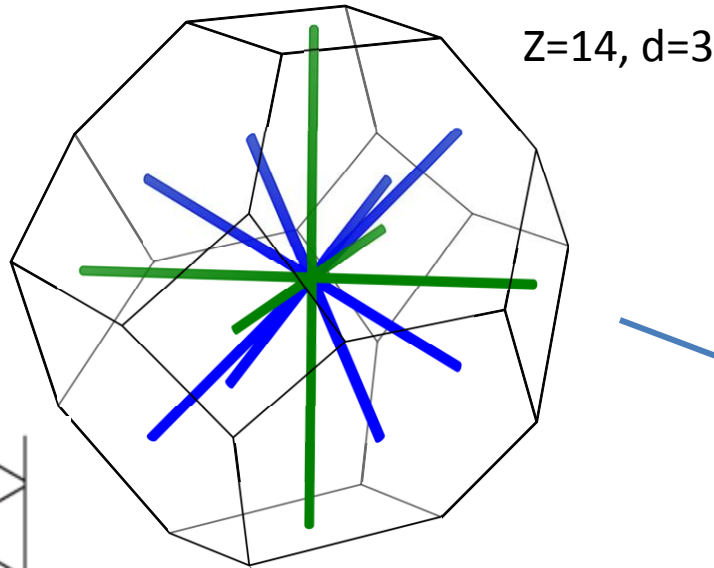
pentamode materials

**pentamode lattice: statics**

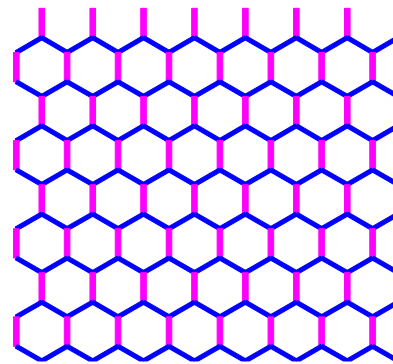
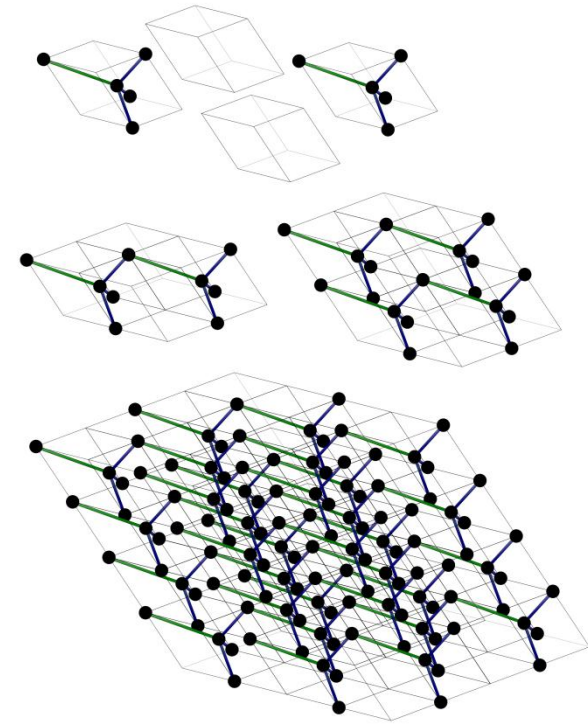
pentamode lattice: dynamics

# stretch dominated effective elastic moduli

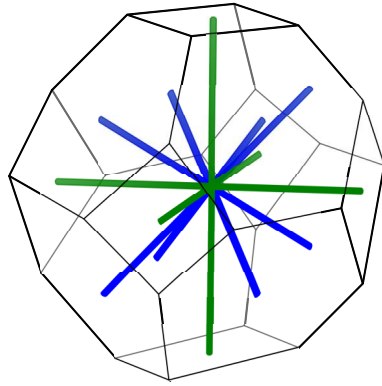
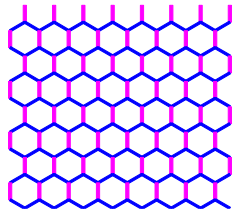
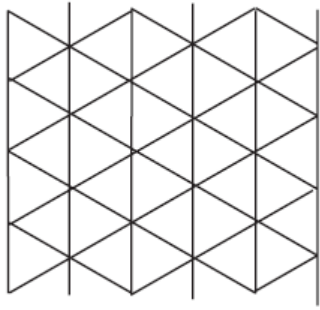
Fully stiff  
 $Z > 11, d=3$   
 $Z > 5, d=2$



Pentamodal  
 $Z=d+1$







Ingredients:

$Z$  = coordination #

$R_i, \mathbf{e}_i$  length, direction

$M_i = \int_0^{R_i} \frac{dx}{E_i A_i}$  axial compliance

$V$  = unit cell volume

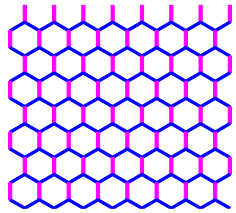
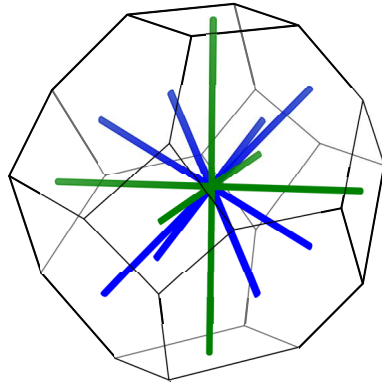
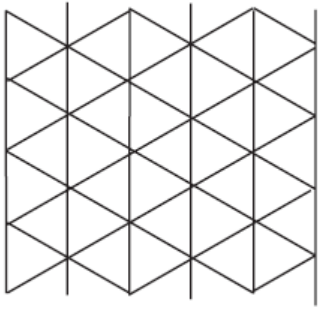
## Effective elastic moduli

$$\mathbf{C} = \frac{1}{V} \sum_{i,j=1}^Z R_i R_j \sqrt{M_i M_j} P_{ij} (\mathbf{v}_i \otimes \mathbf{v}_i) \otimes (\mathbf{v}_j \otimes \mathbf{v}_j) \quad \text{where}$$

$$P_{ij} = \delta_{ij} - \mathbf{v}_i \cdot \left( \sum_{k=1}^Z \mathbf{v}_k \otimes \mathbf{v}_k \right)^{-1} \cdot \mathbf{v}_j, \quad \mathbf{v}_i \equiv M_i^{-1/2} \mathbf{e}_i$$

## Effective elastic moduli of stretch dominated lattices

Ingredients:



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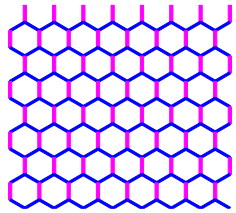
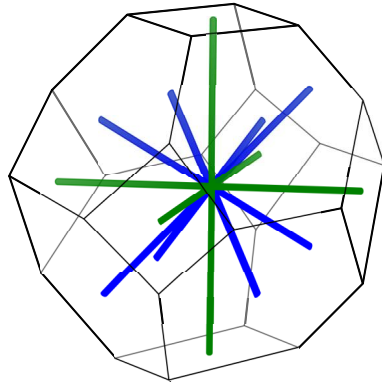
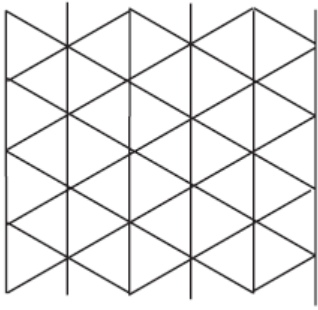
$\mathbf{P}$  is a projector  $\mathbf{P}^2 = \mathbf{P}, \quad \text{tr} \mathbf{P} = Z - d,$



$$\text{rank } \mathbf{C} \leq Z - d$$

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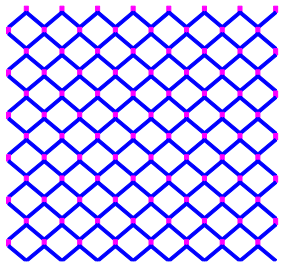


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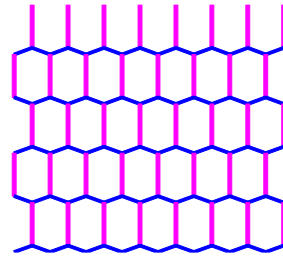
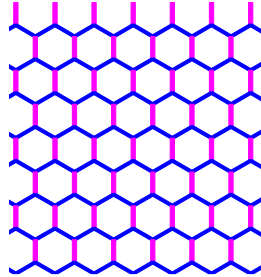
**Pentamode:  $Z=d+1$**

$$\text{rank } \mathbf{C} = 1$$

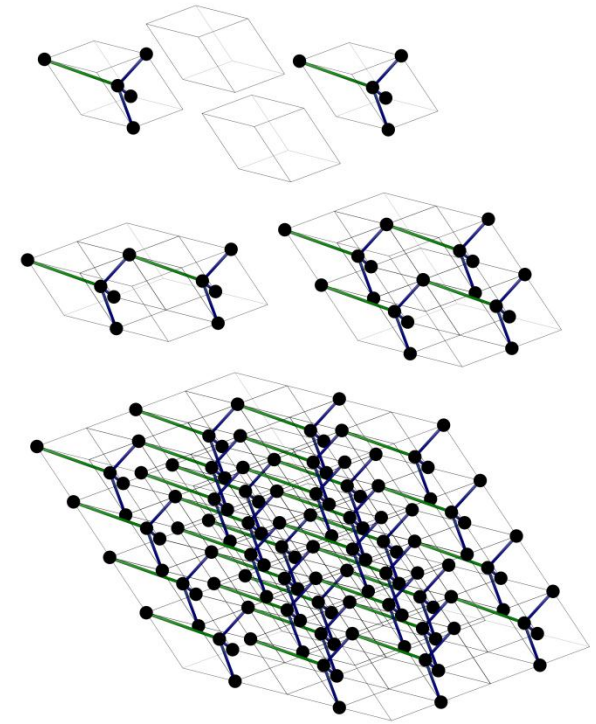
## Explicit static PM moduli for lattices with $Z=d+1$



$Z=3, d=2$



2D PM lattices (all isotropic)



$Z=4, d=3$

Pentamode:  $\mathbf{C} = \mathbf{K} \mathbf{S} \otimes \mathbf{S}$

$$\mathbf{C} = \left( V \sum_{k=1}^{d+1} \gamma_k \right)^{-1} \left( \sum_{i=1}^{d+1} \gamma_i \mathbf{e}_i \otimes \mathbf{e}_i \right) \otimes \left( \sum_{j=1}^{d+1} \gamma_j \mathbf{e}_j \otimes \mathbf{e}_j \right)$$

$$\text{where } \gamma_i = \frac{R_i^2}{M_i} - \frac{\mathbf{R}_i}{M_i} \cdot \left( \sum_{k=1}^{d+1} \frac{1}{M_k} \mathbf{e}_k \otimes \mathbf{e}_k \right)^{-1} \cdot \sum_{j=1}^{d+1} \frac{\mathbf{R}_j}{M_j}$$

$R_i, \mathbf{e}_i$  length, direction,  $M_i = \int_0^{R_i} \frac{dx}{E_i A_i}$  axial compliance,  $V = \text{cell volume}$

pentamode materials

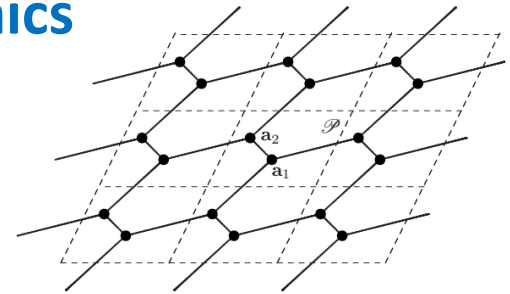
pentamode lattice: statics

**pentamode lattice: dynamics**

# Semi-analytical methods for lattice dynamics

**Colquitt** et al. *Proc. R. Soc. A*, 2011 and 2013.

- 2D
- Longitudinal and flexural waves
- Effective mass underestimated if flexural waves left out



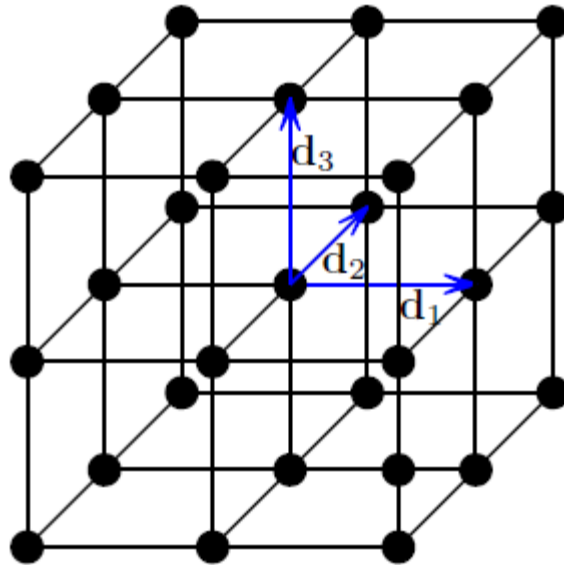
**Leamy**, *J. Sound. Vib.*, 2012

- Wave based approach, 2D, using reflection & transmission

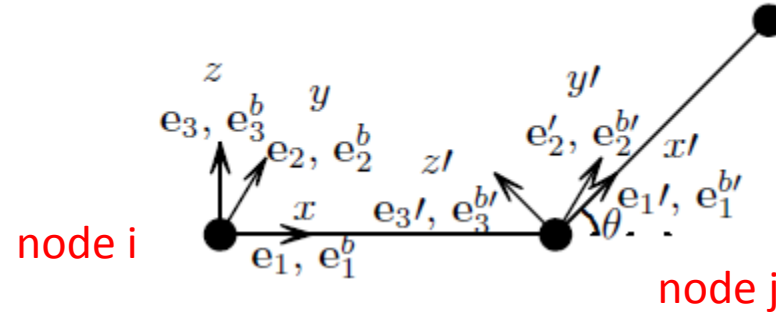
**Here** – 2D and 3D, L and flex waves

- Consistent method 2D, 3D
- Low frequency asymptotics (2D)
- Correct effective mass

Example: cubic lattice



## Lattice dynamics: for each rod



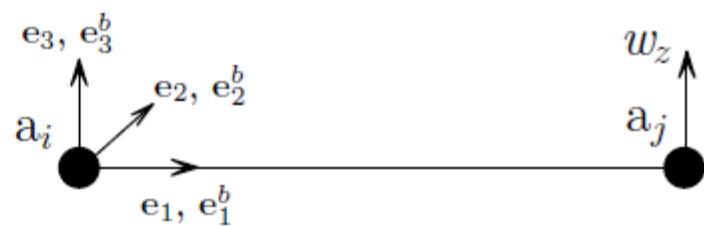
### 1) Longitudinal wave equation

$$\mu_{ij} \frac{\partial^2}{\partial x^2} u_{ij} = -\omega^2 d_{ij} u_{ij}, \quad u_{ij}(0) = \mathbf{e}_1 \cdot \mathbf{u}_i, \quad u_{ij}(l_{ij}) = \mathbf{e}_1 \cdot \mathbf{u}_j$$

$$u_{ij}(x) = \frac{\mathbf{e}_1 \cdot \mathbf{u}_i \sin(s_{ij}\omega(l_{ij} - x)) + \mathbf{e}_1 \cdot \mathbf{u}_j \sin(s_{ij}\omega x)}{\sin(s_{ij}\omega l_{ij})}, \quad s_{ij} = \sqrt{\frac{\rho_{ij}}{\mu_{ij}}}$$

### 2) Flexural wave equation

$$\frac{\partial^4 w}{\partial x^4} - \gamma^4 w = 0, \quad x \in [0, l]$$



Bending in orthogonal directions



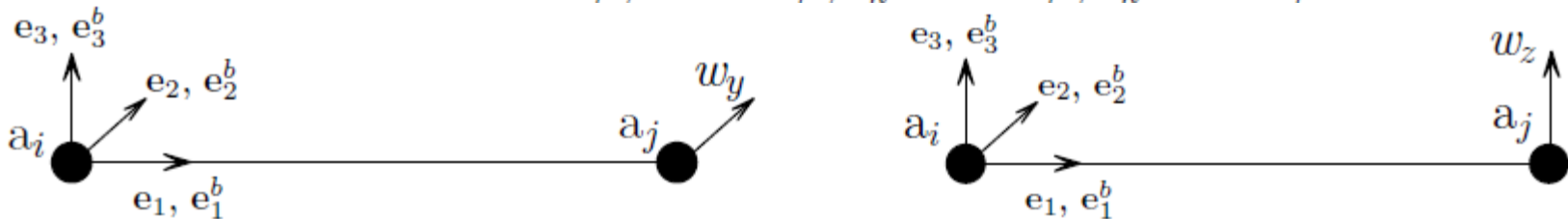
## 2) Flexural wave equation

$$\frac{\partial^4 w}{\partial x^4} - \gamma^4 w = 0, \quad x \in [0, l]$$

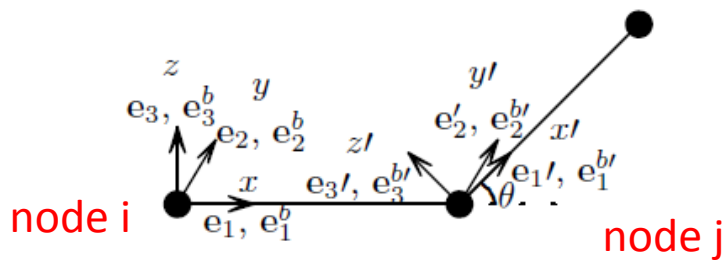
$$\begin{pmatrix} w'''(0) \\ -w''(0) \\ -w'''(l) \\ w''(l) \end{pmatrix} = \mathbf{K}(\omega) \begin{pmatrix} w(0) \\ w'(0) \\ w(l) \\ w'(l) \end{pmatrix}$$

$$\mathbf{K}(\omega) = \frac{1}{1 - c c_h} \begin{pmatrix} \gamma^3(c s_h + s c_h) & \gamma^2 s s_h & -\gamma^3(s + s_h) & \gamma^2(c_h - c) \\ \gamma^2 s s_h & \gamma(s c_h - c s_h) & \gamma^2(c - c_h) & \gamma(s_h - s) \\ -\gamma^3(s + s_h) & \gamma^2(c - c_h) & \gamma^3(c s_h + s c_h) & -\gamma^2 s s_h \\ \gamma^2(c_h - c) & \gamma(s_h - s) & -\gamma^3 s s_h & \gamma(s c_h - c s_h) \end{pmatrix} \quad \mathbf{K} = \begin{pmatrix} \mathbf{K}_1 & \mathbf{K}_2 \\ \mathbf{K}_2^T & \mathbf{K}_3 \end{pmatrix}$$

$$c = \cos \gamma l, \quad s = \sin \gamma l, \quad c_h = \cosh \gamma l, \quad s_h = \sinh \gamma l$$



Bending in orthogonal directions



longitudinal      bending

Total force at point i from rod ij:

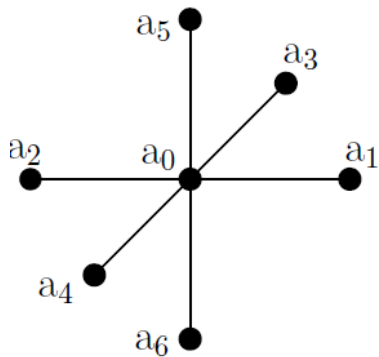
$$\mathbf{f}_{ij} = \mathbf{f}_{ij}^{(1)}(0) + \mathbf{f}_{ij}^{(2)}(0) + \mathbf{f}_{ij}^{(3)}(0)$$

Equilibrium: 
$$\sum_{j \in \mathcal{N}_i} \mathbf{f}_{ij} = -\omega^2 \mathbf{M}_i \mathbf{u}_i, \quad \mathbf{M}_i = \text{diag}(m_i, m_i, m_i, I_i, I_i, I_i)$$

$$\sum_{j \in \mathcal{N}_1} (\mathbf{P}_{1j}^{(2)} \exp(i\mathbf{k} \cdot \mathbf{g}_j) \mathbf{u}_2 - \mathbf{P}_{1j}^{(1)} \mathbf{u}_1) = -\omega^2 \mathbf{M}_1 \mathbf{u}_1$$

$$\sum_{j \in \mathcal{N}_2} (\mathbf{P}_{2j}^{(2)} \exp(i\mathbf{k} \cdot \mathbf{g}_j) \mathbf{u}_1 - \mathbf{P}_{2j}^{(1)} \mathbf{u}_2) = -\omega^2 \mathbf{M}_2 \mathbf{u}_2$$

Floquet conditions



$\mathcal{N}_i$  = set of nodes connected to node i

$$\mathbf{g}_j = \mathbf{a}_j - \mathbf{a}_1, \quad j \in \mathcal{N}_2$$

$$\mathbf{g}_j = \mathbf{a}_j - \mathbf{a}_2, \quad j \in \mathcal{N}_1$$

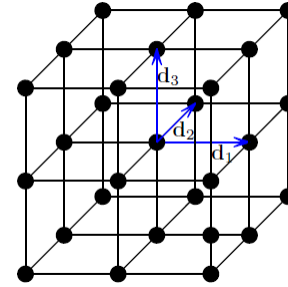
$$\mathbf{P}_{ij}^{(1)} = \tilde{\mu}_{ij} \tilde{s}_{ij} \cot \tilde{s}_{ij} \mathbf{e}_1 \mathbf{e}_1^T + \lambda_{ij} (\mathbf{e}_2, \mathbf{e}_3^b) \mathbf{K}_1 (\mathbf{e}_2, \mathbf{e}_3^b)^T + \lambda_{ij} (\mathbf{e}_3, -\mathbf{e}_2^b) \mathbf{K}_1 (\mathbf{e}_3, -\mathbf{e}_2^b)^T$$

$$\mathbf{P}_{ij}^{(2)} = \tilde{\mu}_{ij} \tilde{s}_{ij} \csc \tilde{s}_{ij} \mathbf{e}_1 \mathbf{e}_1^T - \lambda_{ij} (\mathbf{e}_2, \mathbf{e}_3^b) \mathbf{K}_2 (\mathbf{e}_2, \mathbf{e}_3^b)^T - \lambda_{ij} (\mathbf{e}_3, -\mathbf{e}_2^b) \mathbf{K}_2 (\mathbf{e}_3, -\mathbf{e}_2^b)^T$$

$$\mathbf{K} = \begin{pmatrix} \mathbf{K}_1 & \mathbf{K}_2 \\ \mathbf{K}_2^T & \mathbf{K}_3 \end{pmatrix}$$

## equation/dispersion relation

$$\mathbf{H}\mathbf{u} = \omega^2\mathbf{M}\mathbf{u}$$

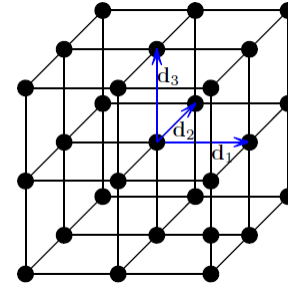


$$\mathbf{u} = \begin{pmatrix} \mathbf{u}_1 \\ \mathbf{u}_2 \end{pmatrix}, \quad \mathbf{M} = \text{diag}(\mathbf{M}_1, \mathbf{M}_2), \quad \mathbf{H} \equiv \mathbf{H}(\omega, \mathbf{k}) = \begin{pmatrix} \mathbf{H}_1 & \mathbf{H}_2 \\ \mathbf{H}_2^+ & \mathbf{H}_3 \end{pmatrix}$$

$$\mathbf{H}_1 = \sum_{j \in \mathcal{N}_1} \mathbf{P}_{1j}^{(1)}, \quad \mathbf{H}_2 = - \sum_{j \in \mathcal{N}_1} \mathbf{P}_{1j}^{(2)} \exp(i\mathbf{k} \cdot \mathbf{g}_j), \quad \mathbf{H}_3 = \sum_{j \in \mathcal{N}_1} \mathbf{P}_{1j}^{(3)}$$

## equation/dispersion relation

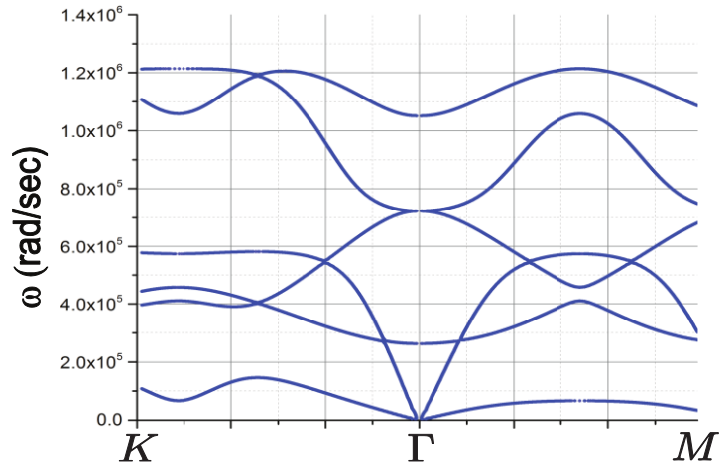
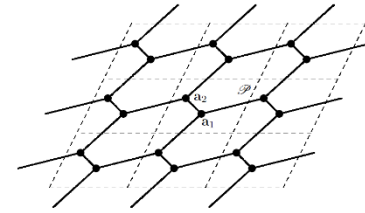
$$\mathbf{H}\mathbf{u} = \omega^2\mathbf{M}\mathbf{u}$$



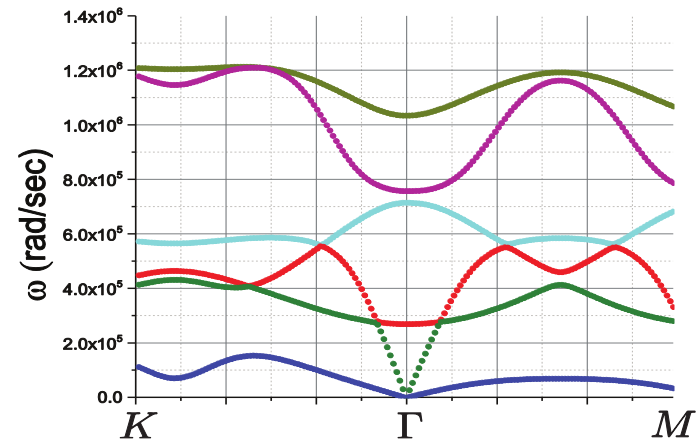
$$\mathbf{u} = \begin{pmatrix} \mathbf{u}_1 \\ \mathbf{u}_2 \end{pmatrix}, \quad \mathbf{M} = \text{diag}(\mathbf{M}_1, \mathbf{M}_2), \quad \mathbf{H} \equiv \mathbf{H}(\omega, \mathbf{k}) = \begin{pmatrix} \mathbf{H}_1 & \mathbf{H}_2 \\ \mathbf{H}_2^+ & \mathbf{H}_3 \end{pmatrix}$$

$$\mathbf{H}_1 = \sum_{j \in \mathcal{N}_1} \mathbf{P}_{1j}^{(1)}, \quad \mathbf{H}_2 = - \sum_{j \in \mathcal{N}_1} \mathbf{P}_{1j}^{(2)} \exp(i\mathbf{k} \cdot \mathbf{g}_j), \quad \mathbf{H}_3 = \sum_{j \in \mathcal{N}_1} \mathbf{P}_{1j}^{(3)}$$

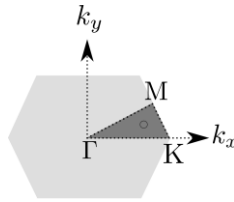
# Honeycomb: 2D pentamode



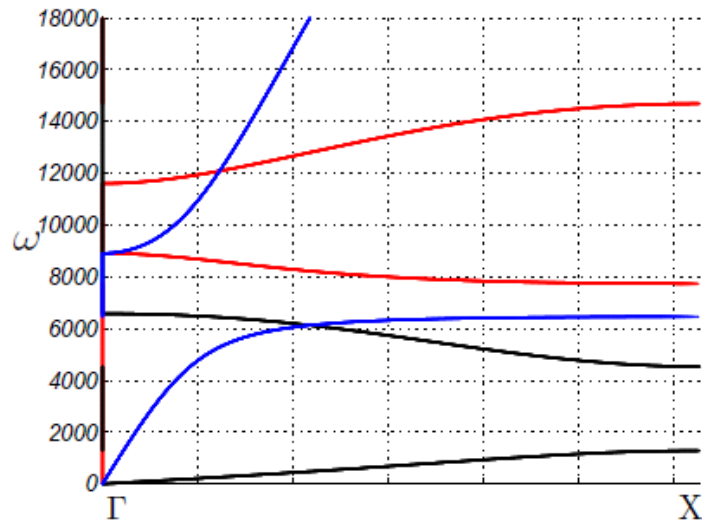
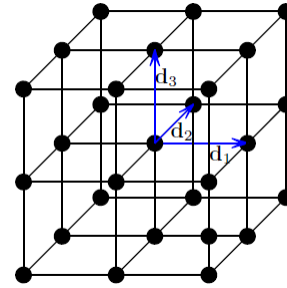
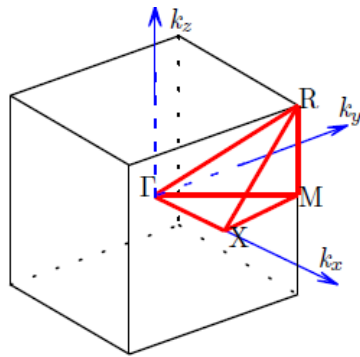
Beam theory



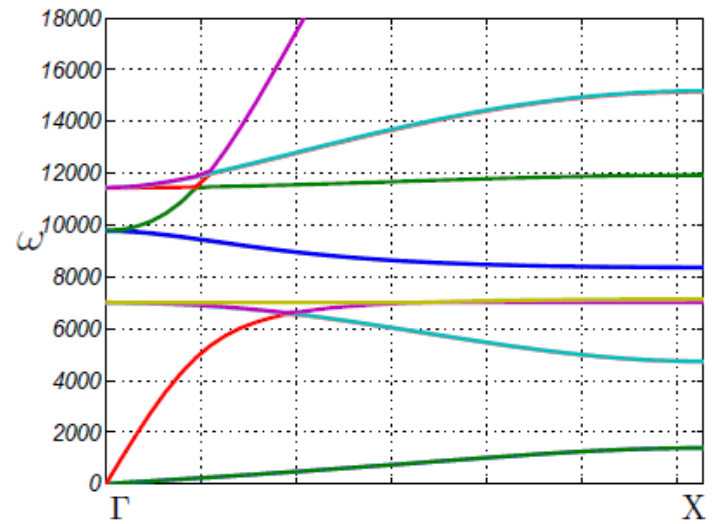
COMSOL



Aluminum beams      length : thickness = 12.5 : 1



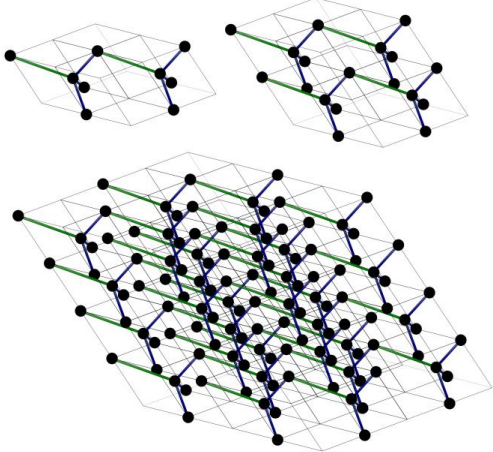
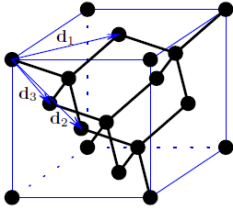
Beam theory



COMSOL

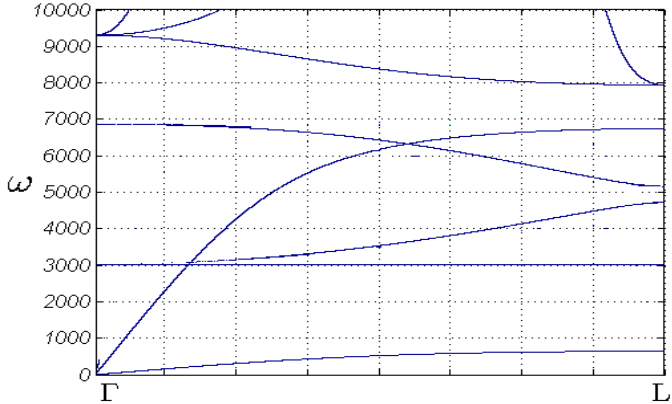
Aluminum beams      length : thickness = 20 : 1

# Diamond lattice : 3D pentamode

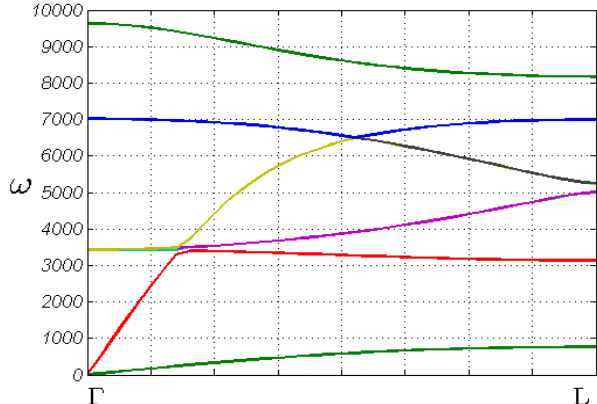


steel rods

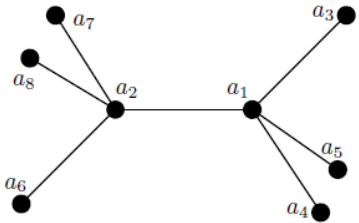
length : thickness = 20 : 1



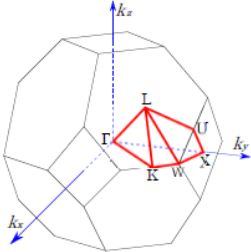
Beam theory



COMSOL

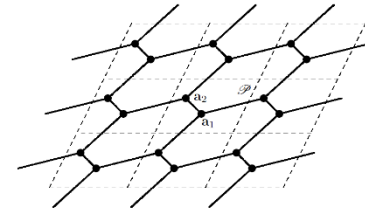


(A) Diamond lattice



(B) IBZ

## Low frequency asymptotics : 2D



Gives correct effective mass:  $m = m_1 + m_2 + \sum_{j \in \mathcal{N}_1} \rho_{1j} l_{1j}$

Gives correct quasistatic wave speeds  $c_T^2 = \frac{3l}{2m} \left( \frac{1}{\mu} + \frac{l^2}{12\lambda} \right)^{-1}$ ,  $c_L^2 = c_T^2 + \frac{3l}{4m} \mu$



# Challenges

## dynamics:

$$\mathbf{H}\mathbf{u} = \omega^2 \mathbf{M}\mathbf{u} \quad \mathbf{u} = \begin{pmatrix} \mathbf{u}_1 \\ \mathbf{u}_2 \end{pmatrix}, \quad \mathbf{M} = \text{diag}(\mathbf{M}_1, \mathbf{M}_2), \quad \mathbf{H} \equiv \mathbf{H}(\omega, \mathbf{k}) = \begin{pmatrix} \mathbf{H}_1 & \mathbf{H}_2 \\ \mathbf{H}_2^+ & \mathbf{H}_3 \end{pmatrix}$$

- low frequency asymptotics for 3D
- use as semi-analytical tool, e.g. relate to “dynamics homogenization”, Willis equations (Norris et al., *PRSA* 2012)

## statics

$$\mathbf{C} = \frac{1}{V} \sum_{i,j=1}^Z R_i R_j \sqrt{M_i M_j} P_{ij} (\mathbf{v}_i \otimes \mathbf{v}_i) \otimes (\mathbf{v}_j \otimes \mathbf{v}_j) \quad \text{where}$$
$$P_{ij} = \delta_{ij} - \mathbf{v}_i \cdot \left( \sum_{k=1}^Z \mathbf{v}_k \otimes \mathbf{v}_k \right)^{-1} \cdot \mathbf{v}_j, \quad \mathbf{v}_i \equiv M_i^{-1/2} \mathbf{e}_i$$

- stretch + **bending**
- relate to asymptotics of dynamic model

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Weidlinger  
U. Bordeaux 1

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and to you



for listening!